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# Parametric Design of A MEMS Actuated Nanolaminate Deformable Mirror

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## **Parametric and Optimal Design: An organized way to design systems**

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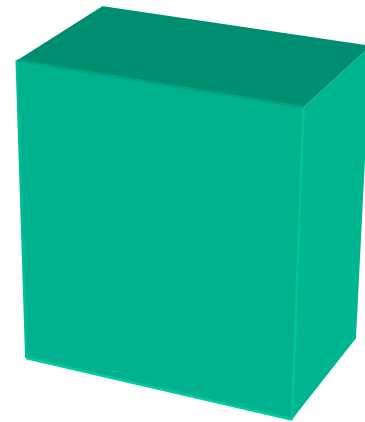
- Come up with a conceptual architecture
- Determine goals of design
- Create a parametric model
  - Determine equations that describe important aspects of design in terms of parameters
- Use model to determine optimal design parameters



## Parametric and Optimal Design Example: Define Problem

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- Project: make a box to hold stuff
- Goals:
  - Volume of box must be  $V$
  - Must use as little cardboard as possible
  - Must be rectangular

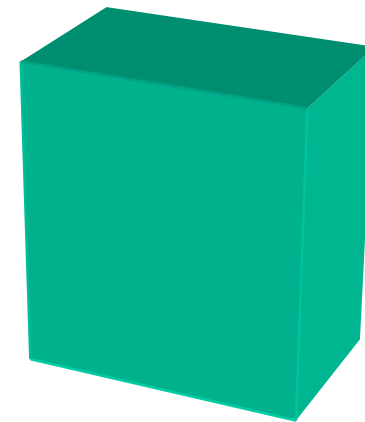




## Parametric and Optimal Design Example: Determine Architecture

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- Architecture: Rectangular prism
  - All six sides made of rectangular pieces of cardboard
- Meets requirements:
  - Holds stuff
  - Rectangular

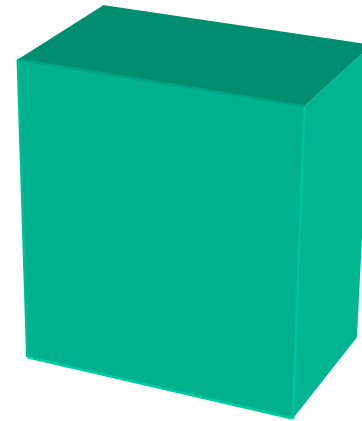




## Parametric and Optimal Design Example: Determine Goals

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- Goals:
  - Volume of box is  $V$
  - Surface area of box must be minimized





## Parametric and Optimal Design Example: Create Parametric Model

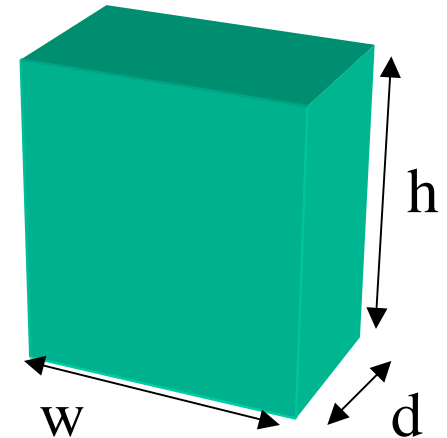
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$$V = w \times h \times d$$

$$A = 2(w \times h + w \times d + d \times h)$$

Which simplifies to:

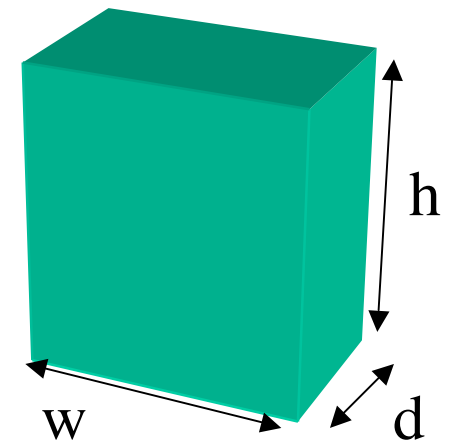
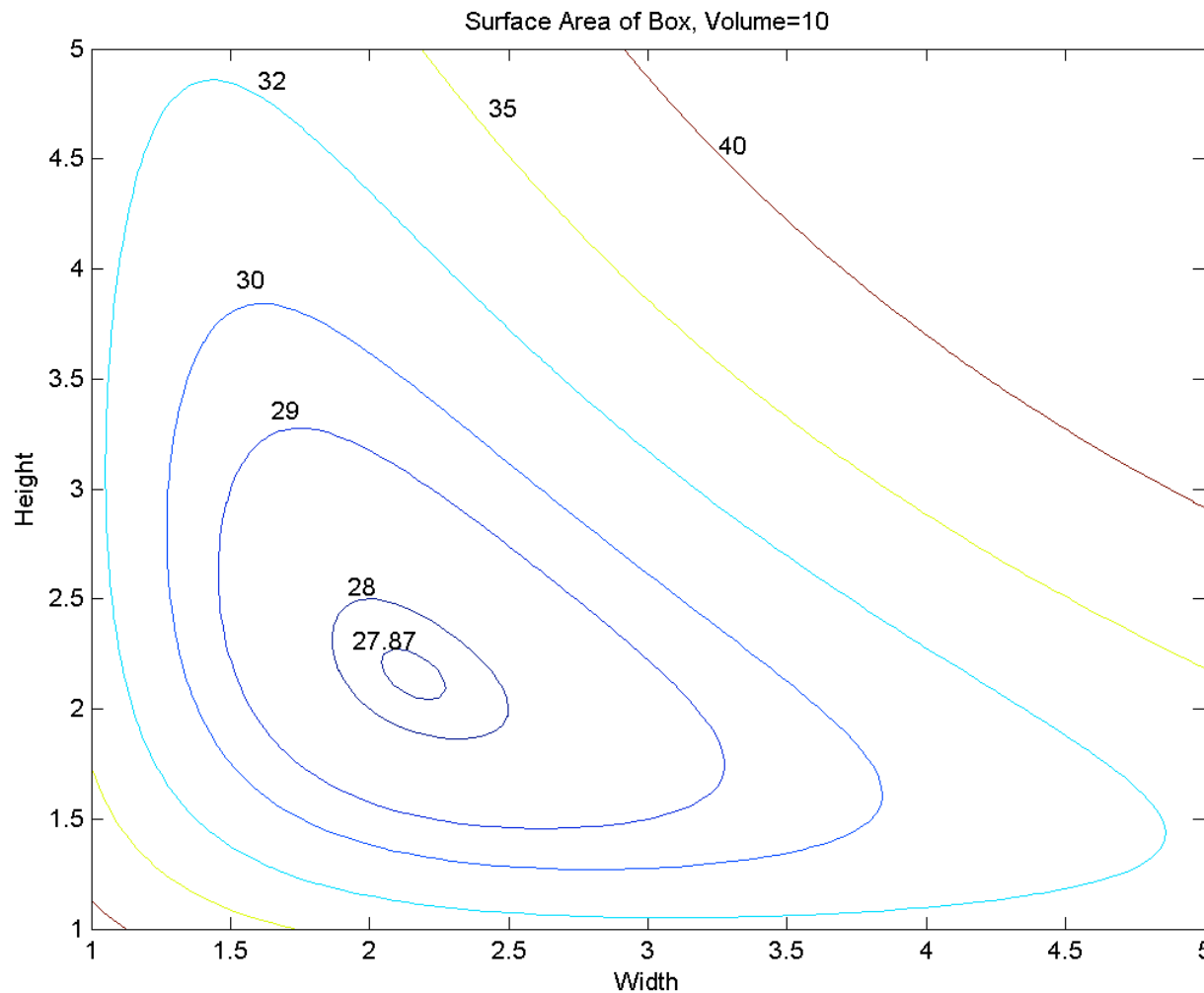
$$A = 2 \left( w \times h + \frac{V}{h} + \frac{V}{w} \right)$$





# Parametric and Optimal Design Example: Use Mathematical Model to Determine Solution

$$\text{Solution: } w = h = d = \sqrt[3]{V}$$

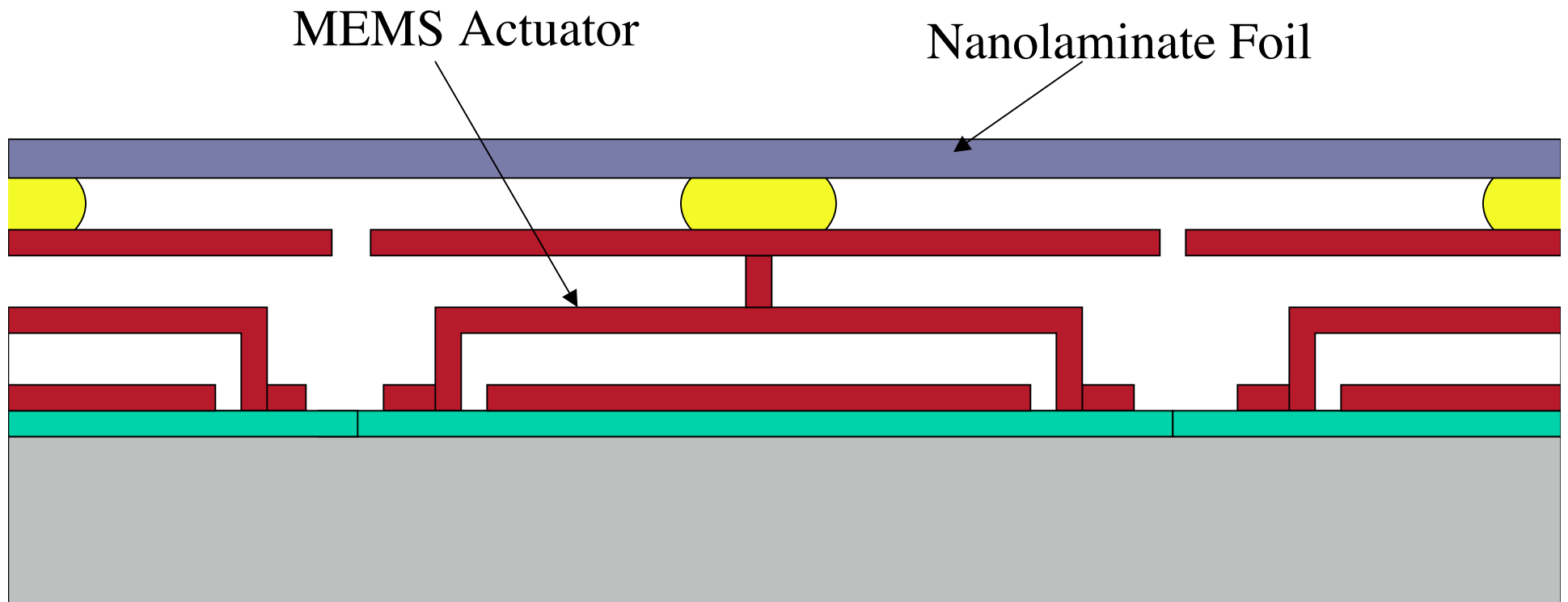




## System Architecture

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- Architecture
  - NL foil bonded to electrostatic actuator







## **Design Goals**

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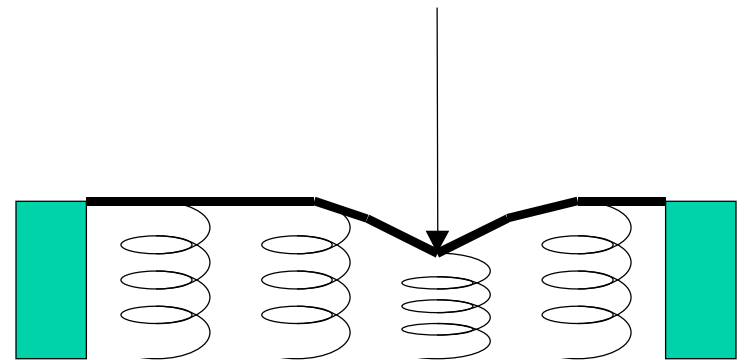
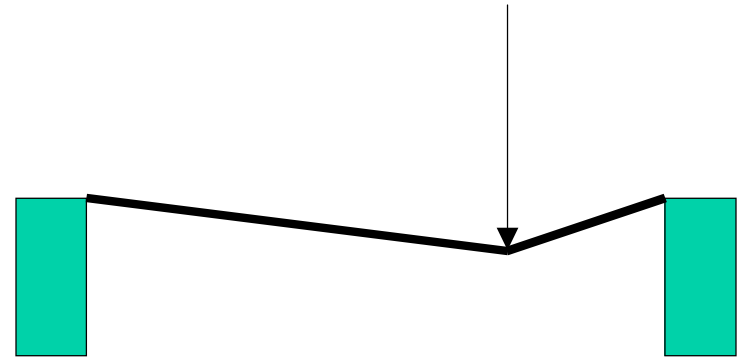
- Applications will specify
  - Maximum voltage allowable
  - Minimum natural frequency
  - Minimum value of maximum displacement
  - Pixel size
  - Minimum cross-talk or minimum spatial frequency



## Design Goals: Cross-talk / Spatial Frequency

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- NL foil stiffer than MEMS actuator
  - Behaves like a trampoline
  - Lots of cross talk
  - Not capable of high spatial frequency features
- MEMS actuator stiffer than NL foil
  - Behaves like a mattress
  - Low Cross Talk
  - Capable of High Spatial Frequency features

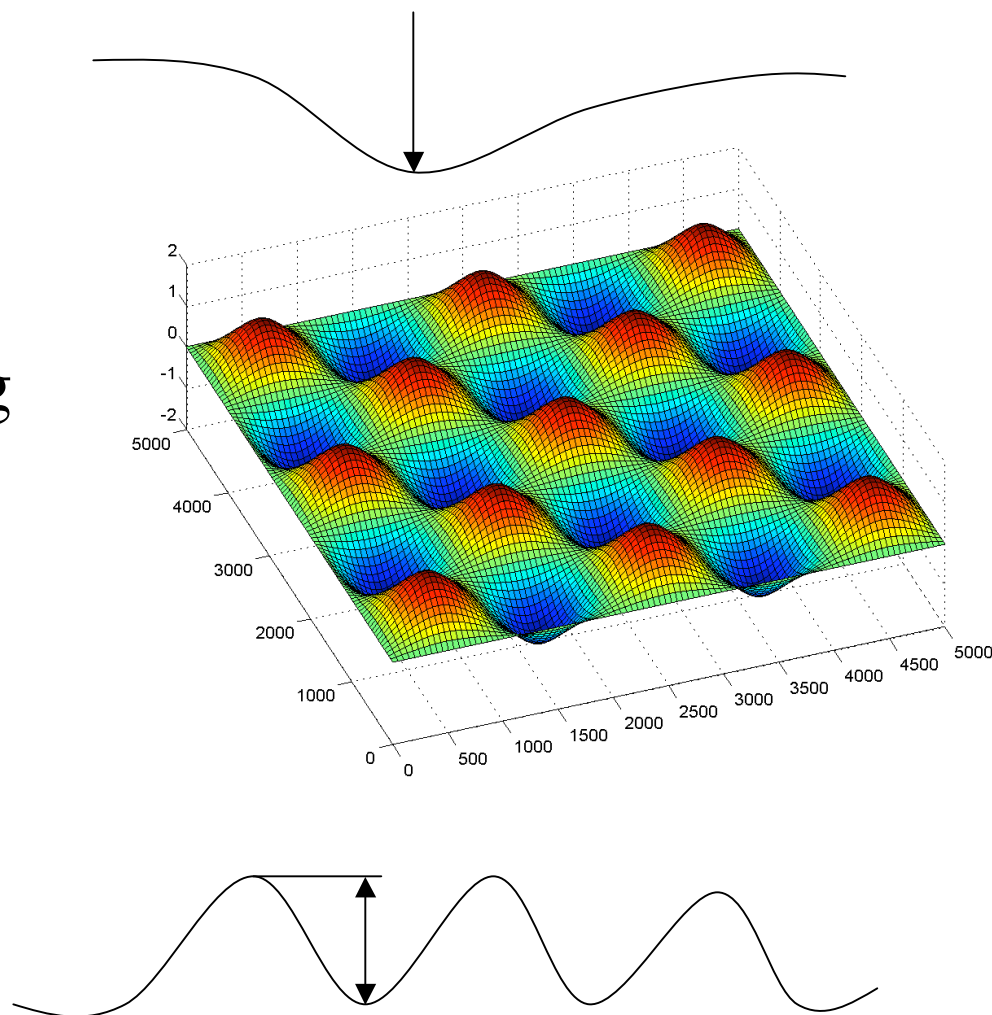




## Design Goals:

### Checker board pattern, a rational figure of merit

- Single point load
  - Difficult to model
  - Difficult to define figure of merit
- Checkerboard loading
  - Every other pixel actuated
  - Obvious figure of merit: Difference in displacement of adjacent pixels





## Revised Design Goals:

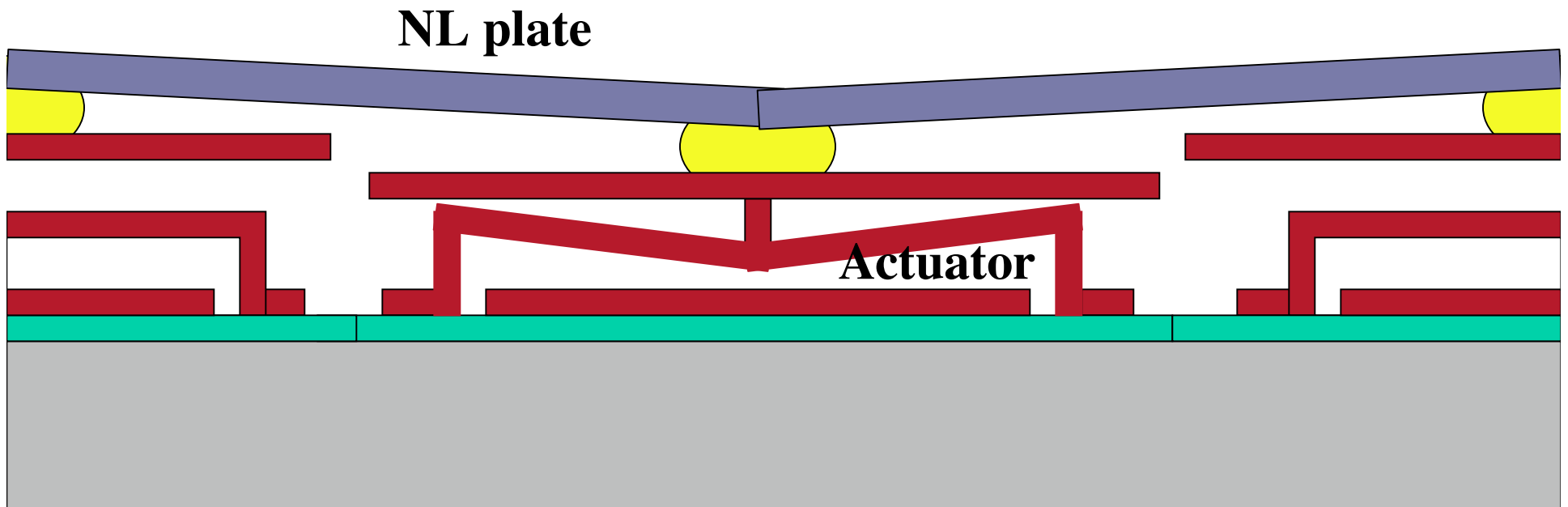
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- Applications will specify
  - Maximum voltage allowable
  - Minimum natural frequency
  - Minimum value of maximum **displacement of actuated pixel-maximum displacement of adjacent un-actuated pixel**
  - Pixel size
  - Minimum cross-talk or minimum spatial frequency



## Develop Parametric Model

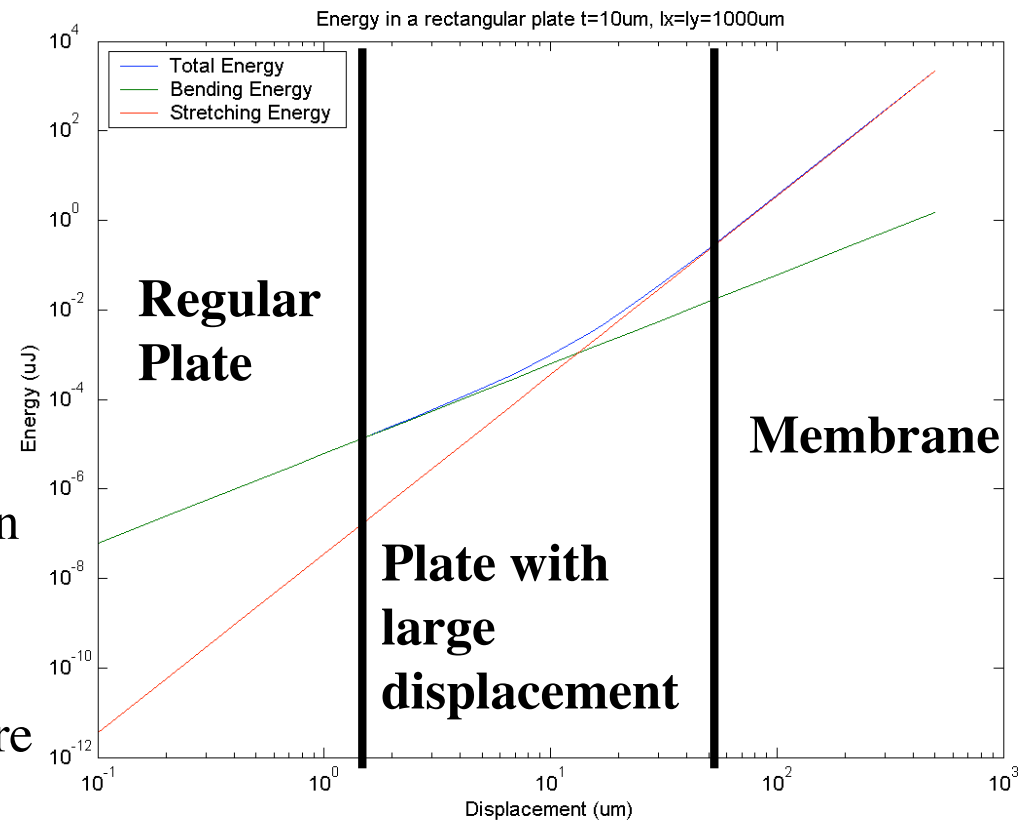
- Parametric Model of Actuator
- Parametric Model of NL foil
- Combine into system model





## Parametric Model: Determine Structure Type

- Regular plate
  - Resistance dominated by bending
  - Displacement less than thickness
- Membrane
  - Resistance dominated by stretching
  - Displacement much larger than thickness
- Plate with large deflections
  - Both bending and stretching are important
  - Displacement on order of thickness





## Parametric Model of Actuator

Differential equation for large deflection plate

$$\nabla^4 z = \frac{1}{D} \left( q(x) + \frac{1}{2} \frac{Eh}{(1-\nu^2)} \left( \left( \frac{\partial z}{\partial x} \right)^2 \frac{\partial^2 z}{\partial x^2} + \left( \frac{\partial z}{\partial y} \right)^2 \frac{\partial^2 z}{\partial y^2} \right) \right)$$

Where

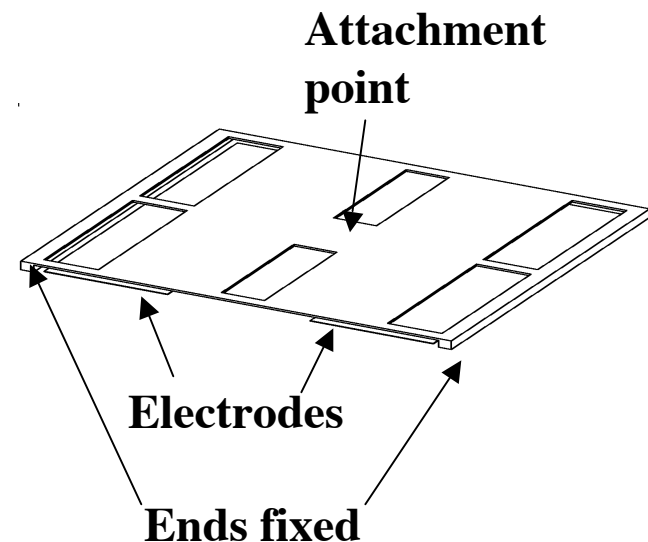
$$D = \frac{Et^3}{(1-\nu^2)} \quad q(x) = \frac{1}{2} \frac{\epsilon_0 V^2}{(g-z)^2} dx dy$$

BMC type actuator only bends in one dimension

$$\nabla^4 z = \frac{1}{D} \left( q(x) + \frac{1}{2} \frac{Et}{(1-\nu^2)} \left( \left( \frac{\partial z}{\partial x} \right)^2 \frac{\partial^2 z}{\partial x^2} \right) \right)$$

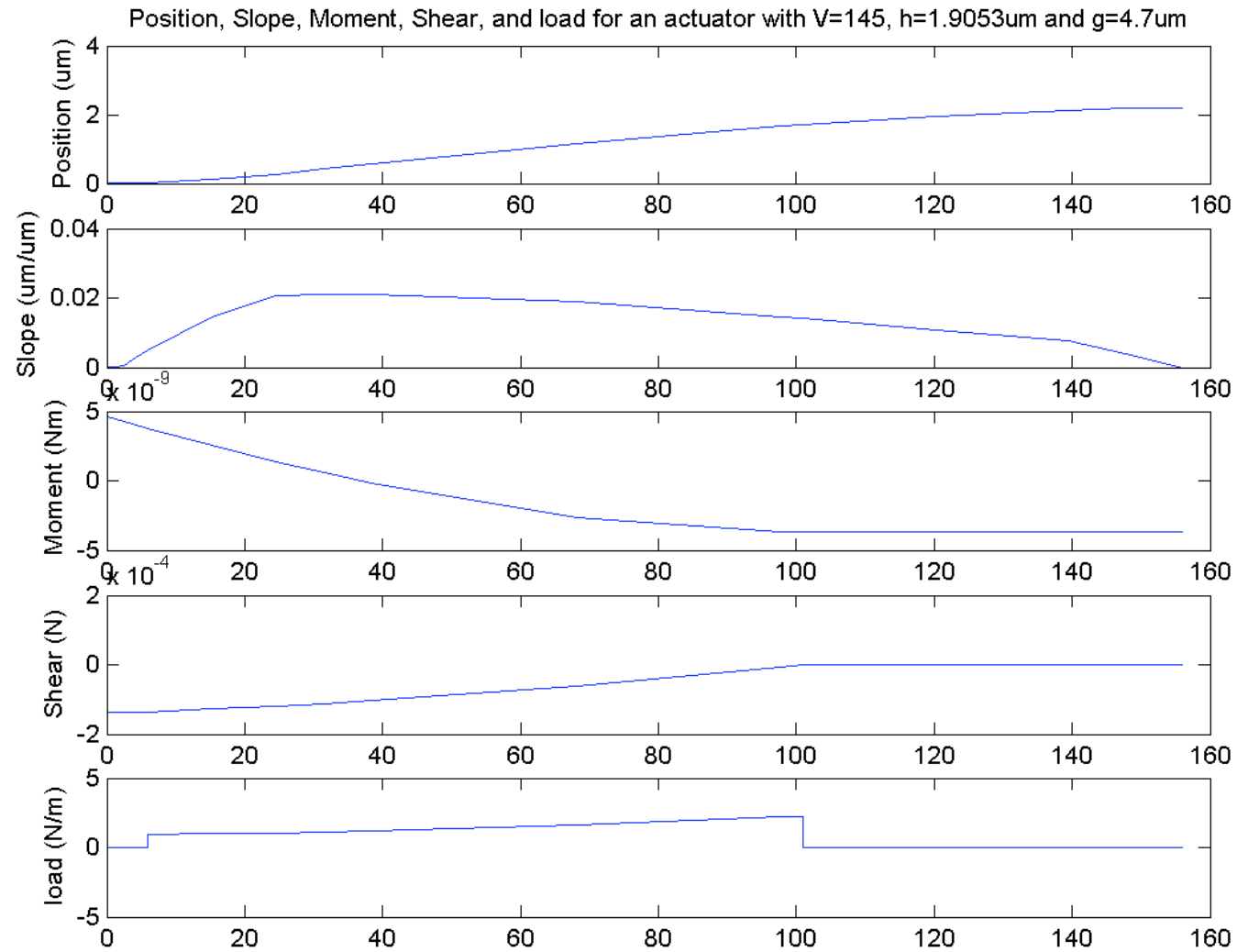
Solve with ODE solver

### BMC actuator





# Results of Actuator Model



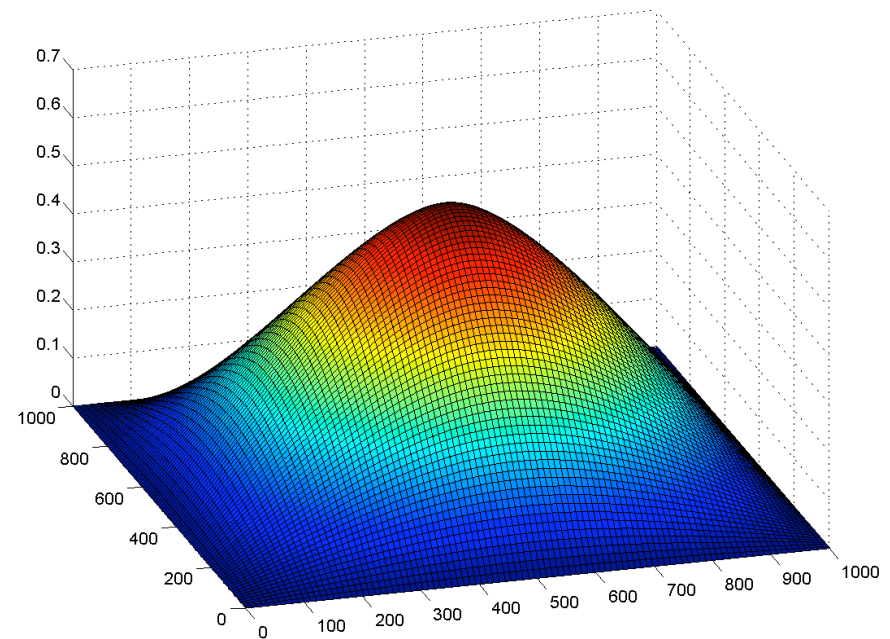




## Parametric Model of NL foil: Solution for Plates with Large Displacements

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- 2-D case of actuator problem
  - Now a PDE
- Rayleigh-Ritz Solution
  - Solve for small displacement to determine shape
  - Solve for total energy to determine large displacement behavior





## Parametric Model of NL foil: Navier Solution- Regular plates

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- Equation for small displacements  $\nabla^4 z = \frac{1}{D} \left( q(x) + \frac{1}{2} \frac{Eh}{(1-\nu^2)} \left( \left( \frac{\partial z}{\partial x} \right)^2 + \frac{\partial^2 z}{\partial x^2} \right) + \left( \frac{\partial z}{\partial y} \right)^2 + \frac{\partial^2 z}{\partial y^2} \right)$

- Assume load is sum of trigonometric functions

$$q(x, y) = \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} a_{mn} \sin\left(\frac{m\pi x}{l_x}\right) \sin\left(\frac{n\pi y}{l_y}\right)$$

- In this case a point load for each actuator

- Displacement will also be a sum of trigonometric functions

$$z(x, y) = \frac{1}{\pi^4 D} \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} \left( \frac{a_{mn}}{l_x^2 + l_y^2} \right) \sin\left(\frac{m\pi x}{l_x}\right) \sin\left(\frac{n\pi y}{l_y}\right)$$



## Parametric Model of NL foil: Solve for Energy in Large Displacements

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Solve for energy in plate

$$dU_{bending} = \frac{1}{2} D \left\{ \frac{\partial^2 z}{\partial x^2} + \frac{\partial^2 z}{\partial y^2} \right\} - 2(1-\nu) \left[ \frac{\partial^2 z}{\partial x^2} \frac{\partial^2 z}{\partial y^2} - \left( \frac{\partial^2 z}{\partial x \partial y} \right)^2 \right] dx dy$$

$$dU_{stretching} = \frac{Et}{8(1-\nu^2)} \left[ \left( \frac{\partial z}{\partial x} \right)^4 + \left( \frac{\partial z}{\partial y} \right)^4 + 2\nu \left( \frac{\partial z}{\partial x} \right)^2 \left( \frac{\partial z}{\partial y} \right)^2 \right] dx dy$$

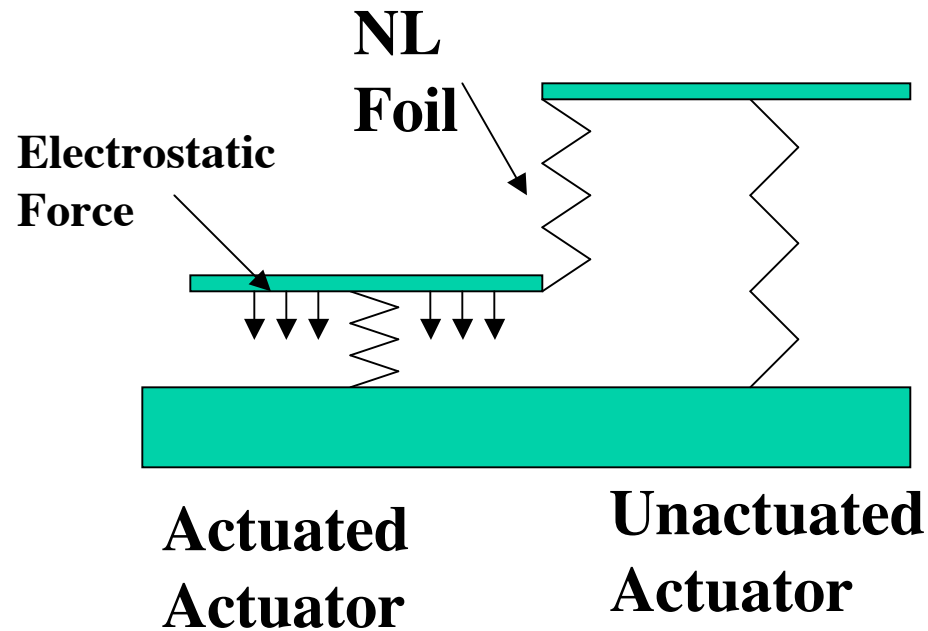
Use total energy to find restoring force



## Parametric Model of System: Combine Actuator and NL models

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- Describe DM as a system of springs





## Natural Frequency of System

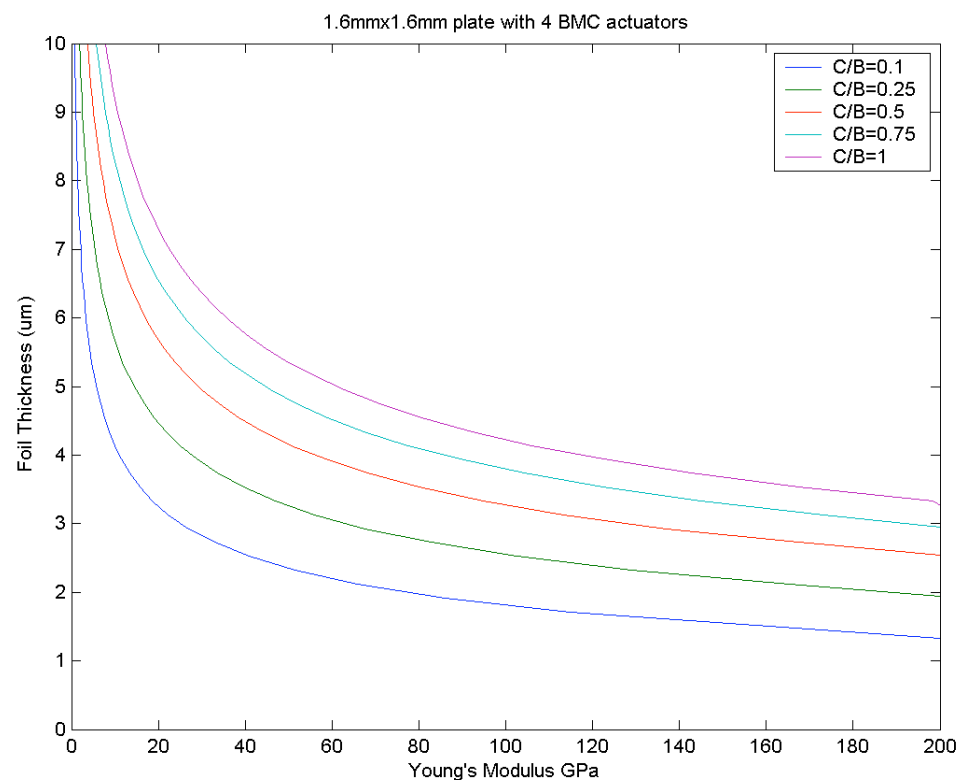
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- Rayleigh method
  - Max potential energy equals max kinetic energy
- Solve for potential energy of both foil and actuator at max displacement
- Find kinetic energy of foil and actuator as a function of natural frequency
- Solve for natural frequency where maximum potential and kinetic energies are equal



## Determine design: Finding a NL Foil to Work With an Existing Actuator

- Actuator exists
  - All actuator parameters already defined
- Must determine NL that allows sufficient deformation





## **Determine design:**

### **Find an Optimal Actuator and Foil combination**

- Determine constraints
  - Maximum and minimum dimensions
  - Maximum Voltage?
  - Max Displacement?
  - Minimum Natural frequency?
- Determine objective function
  - Voltage?
  - 1/Displacement?
- Use minimization software to find parameters that minimize objective function while preserving constraints